## Update to Designing, Operating, and Maintaining Piping System Using PVC

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The use of PVC in the irrigation industry is extensive, but there are some potential problems in designing PVC pipelines if the designer does not give due consideration to damage pressure changes can cause on the pipelines. Modern irrigation systems use electronic controls for turning valves on and off, and in doing so, pressure surges can occur.

Bliesner (Bliesner, 1987) lists several causes of pressure surges in irrigation systems. The most common is due to valve closure. As a valve closes, the kinetic energy of the moving water is converted to pressure in a phenomenon known as water hammer. Most valves are designed to close slowly— perhaps as long as 10 seconds. However, most valves are closed by porting water into the bonnet, and the rate of closure depends on the pressure drop through the valve. The closer the valve is to closing, the faster it closes. Bliesner suggests a rule of thumb that 75% of the fluid flow stoppage occurs in the last 25% of the closure time. That is one of the reasons that the recommended maximum design velocity in the lines is 5 fps.

Air entrapped in system lines can cause significant pressure surge. The flow rate through a restriction—a nozzle for example—depends upon the density of the fluid. (The flow rate is inversely proportional to the square root of the density). Because water is about 100 times as dense as air in pressurized systems, the flow rate of air (and hence the velocity) through a nozzle is about 10 times that of water.<sup>1</sup> As a pocket of air passes through a restriction, the flow rate will accelerate to the flow rate of the air, but when water hits the restriction, the rate must slow to the flow rate of water. This sudden change in velocity results in significant surges in pressure. A reflected wave results from the sudden velocity change, and the wave results in a pressure surge. The following equations have been well-established for calculating the pressure surge. (for a more complete discussion, refer to a hydraulics textbook).

$$a = \frac{4660}{\sqrt{1 + \frac{KD_i}{Et}}}$$

<sup>&</sup>lt;sup>1</sup> Bliesner wrote, "Water is about 5 times more dense than air at 100 psi... when water reaches the opening, the velocity suddenly decreases, since air escapes about 5 times faster than water at 100 psi". That statement is incorrect, but the point is the same. Air can cause water hammer.

Where:a = wave velocity, ft/sK = fluid bulk modulus, (300,000 psi for water)Di = pipe inside diameter, inE = modulus of elasticity of the pipe, (400,000 psi for PVC)t = pipe wall thickness, in.

In plastic pipe, DR, dimension ratio is generally used. DR is the outside diameter ( $D_o$ ) over the thickness, and  $D_o = D_i + 2t$ , so the equation becomes:

$$a = \frac{4660}{\sqrt{1 + \frac{k}{E}(DR - 2)}}$$

The maximum pressure surge is then given by:

$$P = \frac{a (\Delta V)}{2.31 g}$$

Where

ΔV = maximum velocity change, ft/s
g = acceleration due to gravity, 32.2 ft/s<sup>2</sup>
P = pressure surge, psi

As an example, if a flow of 5.0 ft/s is suddenly stopped in a 6 inch class 150 (DR 18) PVC pipe, the surge pressure would be as follows.

$$a = \frac{4660}{\sqrt{1 + \frac{30,000}{40,000}(18 - 2)}}$$

Then,

$$P = \frac{1292 (5.0)}{2.31 (32.2)} = 87 \ psi$$

With this methodology, a table of pressure surges for each dimension ration can be developed. Table 1 below shows the values for one foot per second. Values for higher velocities can be obtained by multiplying the one foot per second value by the velocity.

Dimension Ratio	Pressure Surge, psi
51	10.8
41	11.4
32.5	12.8
26	14.4
25	14.7
21	16.0
18	17.4
14	19.8
13.5	20.2

Table 1. Pressure surges from instantaneous flow stoppage for one foot per second

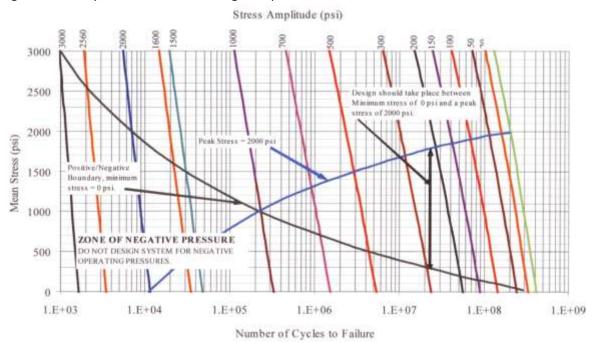
The LASCO fittings web page has a handy water hammer and surge pressures calculator. The inputs are system pressure, velocity change, pipe run length, pipe material, pipe diameter, and SDR. It returns the wave velocity, critical valve close time, surge pressure and total spike. The URL is: <a href="http://www.lascofittings.com/supportcenter/surgeCalc.asp">http://www.lascofittings.com/supportcenter/surgeCalc.asp</a>. A screen capture for 26 SDR is shown below. According to Moser and Folkman (2008), a good rule of thumb for surge pressure rise in pressure rated pipe is 16 psi per foot per second of velocity change.

## Figure 1. LASCO water hammer on-line calculator

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	Fipe Schedule/SDF	C. OUN 2	u (*	E - Modulus of elasticity, in tension, of the pipe materials		
	Calculate Res	Print For		400,000 = PVC 150,000 = HDPE		
	Dimension Dation		26	Da - Pipe O.D. + Wall thickness		
	Dimension Ratio: Wave Velocity:	-		$(0/\text{sec}) = A = 4,660 + (1 + (K \times D_0) - E))^{10}$		
	Critical Valve Close	Time:		(sec) To = (2 x L) = A		
	Surge Pressure:			(pu) P = (62.4 x A x V) + 4,635.6		
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## **PVC Pipe Design**

Bliesner reported a design technique based upon work done by Vinson (Vinson, 1981). Uni-Bell funded follow-up research at Utah State University which was done by Jeffrey, Moser, and Folkman [JMF](2004). Vinson's equations were developed assuming fatigue was a function of peak stress alone. Later researchers contested this assumption, and JMF showed that Vinson's work led to very conservative designs. JMF showed that mean stress and stress amplitude together influence PVC fatigue, and they developed a new set of design equations. A design graph relating mean stress, cycles to failure and stress amplitude. That graph is reproduced below.



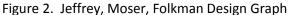


Figure 14. Cyclic design graph

For most irrigation designers, this graph is a bit overwhelming. The process is iterative in that one must first assume a dimension ratio and check it. Then based on the result, a new dimension ratio is assumed. The process continues until there is no change between the assumed and calculated dimension ratio. JMF provided the equations from which the design graph was developed, and those have been solved for typical irrigation design scenarios and are shown in Table 2 below.

F	lecon	nmer	nded	SDR				-	-	-		and n	umb	er of	cycle	es
					IVIa	aximur	n veic		ssure,	er sec psi	ona					
		60	70	80	90	100	110	120	130	140	150	160	170	180	190	200
	0.50	41	41	32.5	32.5	32.5	26	26	25	21	21	21	18	18	18	14
	0.75	41	41	32.5	32.5	26	26	25	21	21	18	18	18	14	14	14
	1.00	41	32.5	32.5	26	26	25	21	21	18	18	18	14	14	14	14
su	2.00	32.5	32.5	26	26	21	21	18	18	14	14	14	14	14		
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	4.00	26	26	21	21	18	18	14	14	14	14					
	5.00	26	26	21	21	18	14	14	14	14						
Cycles,	6.00	26	25	21	18	18	14	14	14	13.5						
Ú	7.00	26	21	21	18	14	14	14	13.5							
	8.00	26	21	18	18	14	14	14								
	9.00	25	21	18	14	14	14	13.5								
	10.00	21	21	18	14	14	14									
		SDR fo	r 0.50 r	nillion	cycles	at 60 p	si is d	ictated	by sho	ort-term	n pressu	ure rati	ing, not	cyclic:	life	

Table 2. PVC SDR selection for assumed operating pressure and lifetime cycle.

Table 2 is used to select the proper SDR given the estimated number of cycles of a system over the lifetime of the system and the operating pressure. For example, if it is anticipated that there will be 2 million cycles over the lifetime of a system with an operating pressure of 100 psi and maximum velocity of 5 fps, the recommended SDR is 21. For the regions shown in red, there are no valid PVC choices. The only case shown where short term pressure governs rather than cyclic pressure governs is 0.5 million cycles at 60 psi. Using cyclic pressure, one could pick SDR of 51, but it will not withstand the pressure surge, so 41 is selected. It will withstand the pressure surge. How the SDR varies with cycles and operating pressure is graphically shown in Figure 2 below.

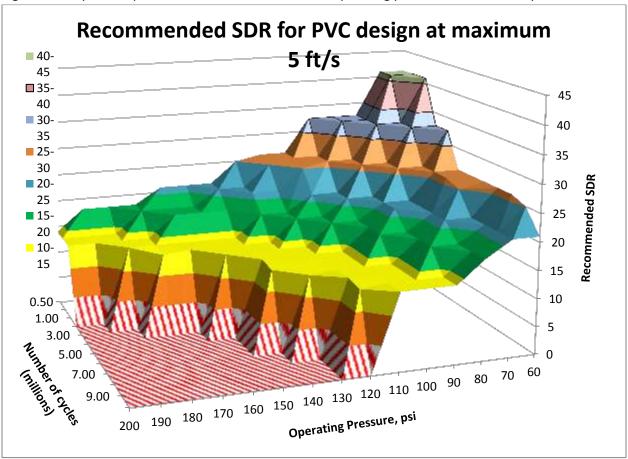


Figure 3. Graphical representation of SDR selection for operating pressure and lifetime cycles.

## **PVC Fitting Design**

For a variety of reasons, PVC fittings are not nearly as durable as is PVC pipe. One reason cited (Edwards, et.al. 1992) is that fittings typically are molded from lower molecular weight material than pipe resulting reduced strength. A second reason is that the shape of the fittings—primarily tees—results in stress concentrations. After testing 6-inch schedule 40 PVC, they concluded that the Vinson design equations were applicable only if the pressures were de-rated for fittings. They proposed de-rating to 60% of rated pressure. Vinson found that the cycles to failure, C, could be found by:

$$C = (5.05 \ x \ 10^{21}) S^{-4.906}$$

where S is the stress, psi.

The stress can be found using the following equation:

$$S = [P(DR - 1)]/2$$

where P is the maximum pressure including the surge, psi, and

DR is the design ratio.

Using a 60% de-rating and substituting  $S_f$  for S, the equation becomes:

$$S_f = \frac{[P(DR-1)]}{2 x (0.60)} = \frac{[P(DR-1)]}{1.2}$$

This de-rating results in reducing the number of cycles the fittings can sustain to about 1/12<sup>th</sup> of that of the pipe. Table 3 below shows the recommended SDR for pressures from 50 to 100 psi and cycles from 500,000 to 3 million. PVC fittings are not recommended for cycling systems with operating pressures in excess of 100 psi.

Table 3. Recommended Pipe Fitting SDR for given number of cycles, operating pressure, and velocity.

Cycles, M		Operating Pressure, psi																
	50		60			70			80			90			100			
0.5	21	18	13.5	18	14	14	18	14	13.5	14	14		14	13.5		14		
1.0	18	14		14	14		14	13.5		14			13.5			13.5		
1.5	18	14		14	13.5		14											
2.0	14	13.5		14			13.5											
2.5	14																	
3.0	14																	

Black is for 3 fts velocity, blue is for 4 ft/s, and red is for 5 ft/s.

References:

A.P. Moser and Steven Folkman, 2008. Buried Pipe Design, 3<sup>thrd</sup> Ed. McGraw-Hill. ISBN 0-07-147689-X. 631 pp.

Ron D. Bliesner, 1987. Designing, Operating and Maintaining Piping Systems using PVC Fittings. The Irrigation Association. Fall Church, VA. 14 pp.

D.B. Edwards, B. Lehman, and R.M. Cohen. 1992. Fatigue testing of PVC Pipe Fittings. Journal of Vinyl Technology, June 1992, Vol. 14, No. 2, pp 69-73.

Uni-Bell, 2005. Handbook of PVC Pipe. Uni-Bell PVC Pipe Association. Dallas, TX. 522 pp.

Jared D. Jeffrey, A.P. Moser, and Steven Folkman. 2004. Long-Term Cyclic Testing of PVC Pipe. Final Report to Uni-Bell PVC Pipe Association. URL: <u>http://igs.nigc.ir/igs/STANDARD/artic/PP-63.pdf</u>. (6/14/2011)